

Anti-Windup and 2DOF Design for Permanent-Magnet Synchronous Motor Speed Controller

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Abstract

A novel speed controller for permanent-magnet synchronous motor (PMSM) is proposed. With the usage of an analytical two-degree-of-freedom (2DOF) proportional integral (PI) controller design method in speed control, good speed-reference tracking and good load-torque rejection can be achieved simultaneously. A “back calculation and tracking” anti-windup (AW) method is used to deal with the “windup” problems. In both simulation and experiment, the proposed designs are shown to exhibit satisfactory performance.

Keywords: *Permanent-magnet synchronous motor (PMSM), Speed control, Two-degree-of-freedom (2DOF), Anti-windup (AW)*

1. Introduction

Permanent-magnet synchronous motor (PMSM) is widely applied in various fields due to its advantages of high efficiency and high power density [1]. For a long time, significant attention is being directed to speed control of PMSM drives. The reasons for this interest are that the dynamics may be nonlinear and/or time varying and that it is essential to gain both good speed-reference tracking and good load-torque rejection performance. The latter two objectives cannot both be met with a standard one-degree-of-freedom (1DOF) proportional integral (PI) controller. Besides, within PMSM control systems, certain constraints are required to protect the motor and the power electronics converter; for instance, restrictions on the magnitude of voltages and currents. These constraints can cause controller “windup” problems when the controller attempts to exceed them and lead to performance degradation. For better performance, there is the requirement to consider these saturation constraints in the design of PMSM control systems, particularly in high-performance applications where saturation is more prevalent.

In view of the limitations of standard 1DOF PI controller, various nonlinear methods, including neural networks, fuzzy logic [2], and sliding mode control, have been proposed. But for the engineering application, most of these methods are too complex. Many publications focus on two-degree-of-freedom (2DOF) PI controller, [3]-[6]. The so-called 2DOF controller is not a fixed pattern, but includes a variety of forms. The basic pattern is the so-called integral proportional (IP) control [3]. Alternative methods include the usage of a phase-locked loop, auto-disturbance rejection, load-torque estimation and feedforward, and adaptive techniques. An analytical 2DOF PI controller design method was proposed in reference [4]. Unlike many of the previously proposed linear controllers, the gains of the controller are analytically parametrized in the inertia as well as in the desired closed-loop bandwidth. But the electrical dynamics is neglected in the design process.

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To deal with the “windup” problems caused by saturation constrains, many anti-windup (AW) methods are proposed. The AW compensator variants include an integrator reset scheme, back calculation and tracking [7], and an optimally synthesized low-order dynamic compensator [8]. Performance of these three AW structures is compared through simulation and experiments in reference [9]. Both of “back calculation and tracking” and “low-order dynamic” AW compensation methods provide an improvement over the reset approach and show very similar performance in simulation despite their differing design routines and implementations. The “back calculation and tracking” approach which is applied widely in electrical drive systems is very simple to implement and easy to tune.

In this paper, we apply 2DOF design method and AW compensation method to a PMSM speed control system. Simulations and experiments will be presented to show the performance of the proposed speed controller.

2. Model of PMSM Speed Control System

2.1. PMSM Model

In dynamic analysis of PMSM, Clark and Park transformations are used to convert the three phase system (*abc*-axis) into two phase rotating system (*d-q*-axis) generally. The dynamic equations of PMSM in *d-q*-axis are as follows [10], [11]:

$$\begin{cases} u_d = R_s i_d + L_d \frac{di_d}{dt} - \omega_r i_q L_q \\ u_q = R_s i_q + L_q \frac{di_q}{dt} + \omega_r i_d L_d + \omega_r \Psi_f \\ T_e = p_0 [\Psi_f i_q + (L_d - L_q) i_d i_q] \\ \frac{1}{p_0} \frac{d\omega_r}{dt} = \frac{1}{J} (T_e - T_l) \end{cases} \quad (1)$$

Where u_d and u_q are stator voltages in *d-q*-axis, i_d and i_q are stator currents in *d-q*-axis, L_d and L_q are inductances in *d-q*-axis, ω_r is the rotor electrical angular speed, R is the stator resistance, Ψ_f is the rotor flux, p_0 is the number of poles, J is the inertia of the motor, T_e and T_l are the electromagnetic torque and the load torque.

2.2. Control System of PMSM

The three phase PMSM speed control system which we desire to apply 2DOF design and AW compensation to is shown in Figure 1. The control system consists of an inner current loop and an outer speed control loop. The inner loop contains two independent PI regulators, used to control the magnitude of the stator current components at *d-q*-axis, namely i_{dq} (i_d and i_q). The speed controller is shown in Figure 2. It generates the *d*-axis and *q*-axis current commands, namely i_{dq_cmd} (i_{d_cmd} and i_{q_cmd}), and passes them to the inner loop. Saturation function $\text{Sat}(\|\cdot\|)$ sets the maximum limit of i_{dq_cmd} . For maximum torque per ampere (MTPA) [12] operation, i_{d_cmd} is calculated by i_{q_cmd} and ω_r , particularly $i_{d_cmd}=0$ for surface mounted PMSM.

The transfer function of the mechanical dynamics of the PMSM is immediately found by taking the Laplace transform of (1). It is simply an integrator

$$G_m(s) = \frac{1}{sJ} \quad (2)$$

For the sake of simplicity, the current loop is equivalent to a first order inertia link [12], so the transfer function of the inner loop is

$$G_{ci}(s) = \frac{1}{\tau_i s + 1} \quad (3)$$

Where τ_i is the time constant of the electrical dynamics, which is usually tens of milliseconds.

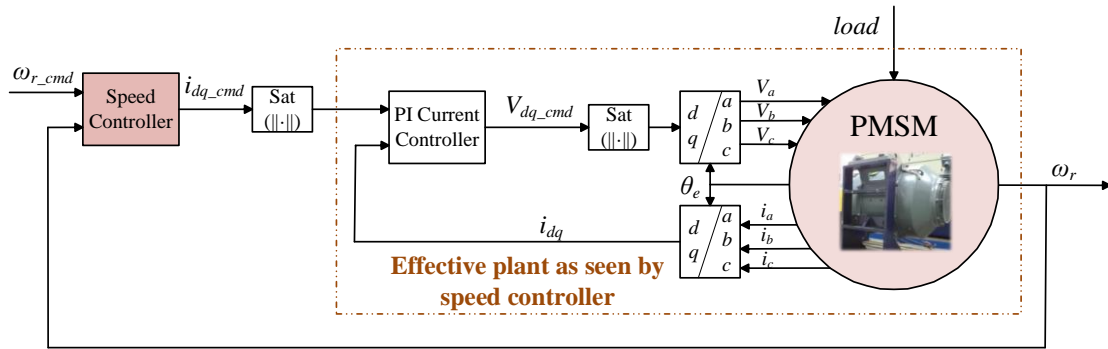


Figure 1. Speed Control System of PMSM

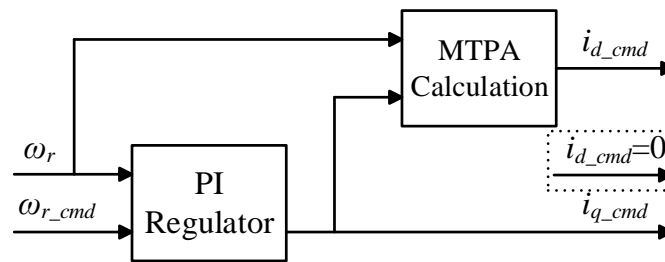


Figure 2. Speed Controller Structure

3. Speed Controller Design

In this section, we introduce an analytical 2DOF PI controller design method and AW compensation method and apply them to the speed controller design for PMSM.

3.1. 2DOF Method

First of all, we analyze the characteristics of the traditional control structure. Block diagram of the standard 1DOF control [3] is shown in Figure 3. Where $X(s)$ is the input, $Y(s)$ is the output, $N(s)$ is the disturbance, $G(s)$ is the controlled object and $F(s)$ is the controller. The close-loop transfer function from $X(s)$ to $Y(s)$ and from $N(s)$ to $Y(s)$ are shown in (4) and (5).

$$G_X(s) = \frac{F(s)G(s)}{1 + F(s)G(s)} \quad (4)$$

$$G_N(s) = -\frac{G(s)}{1 + F(s)G(s)} \quad (5)$$

From (4) and (5) we can see that $G_X(s)$ and $G_N(s)$ are both determined by $F(s)$ only. So systems with 1DOF control cannot adjust the reference tracking and disturbance rejection performance separately.

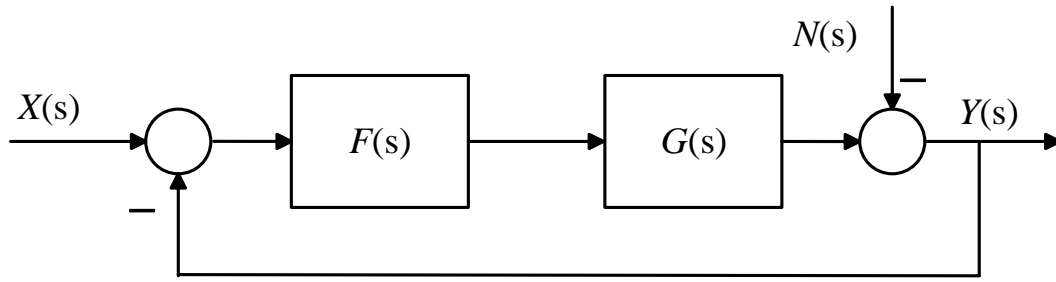


Figure 3. Block Diagram of the 1DOF Control

To make an enhancement of 1DOF control, the structure can be modified by adding a feedforward controller $F_r(s)$ for the reference, as shown in Figure 4

The close-loop transfer function from $X(s)$ to $Y(s)$ is shown in (6) and the close loop transfer function from $N(s)$ to $Y(s)$ is as same as (5).

$$G_x(s) = \frac{F_r(s)G(s) + F(s)G(s)}{1 + F(s)G(s)} \quad (6)$$

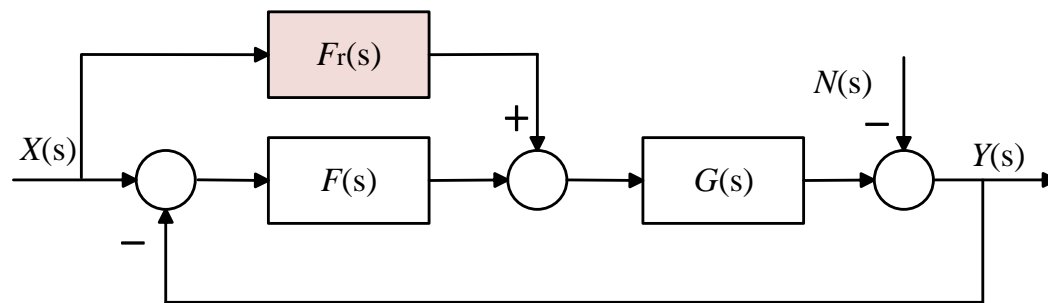


Figure 4. Block Diagram of the 2DOF Control

From (6) and (5) we can see that $G_N(s)$ is determined by $F(s)$ only. While $G_x(s)$ is determined by $F(s)$ and $F_r(s)$ collectively. We can get desired disturbance rejection performance by modifying $F(s)$. And the feedforward controller $F_r(s)$ is act as a patch to improve the reference tracking performance. When the form of $G(s)$ is identified as a goal, the transfer function of $F_r(s)$ can be derived from (6)

$$F_r(s) = F(s)(G_x(s) - 1) + \frac{G_x(s)}{G(s)} \quad (7)$$

3.2. AW Compensation

A signal u^* is defined as the difference of between saturated and unsaturated control signals. And $u^*=0$ indicates the normal operation where $u^* \neq 0$ indicates the existence of a saturation event. In the “back calculation and tracking” method, the AW compensator adopts the form of a scalar feedback gain K_a , which is introduced between the signal u^* and the input to the integrator in the PI controller, as is shown in Figure 5.

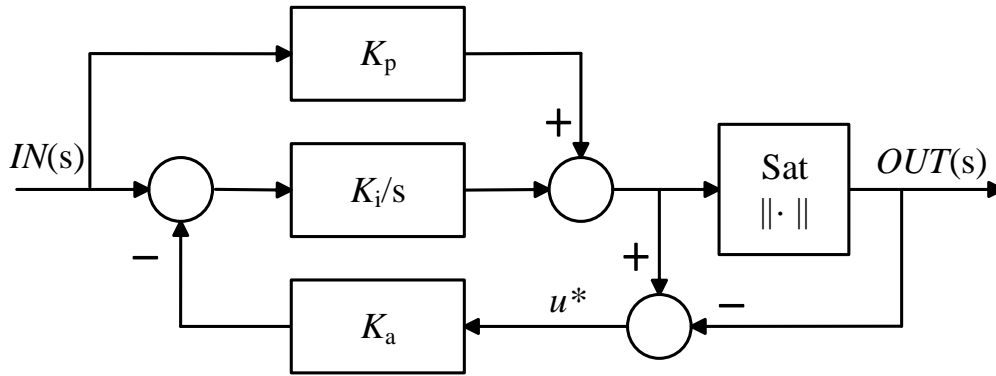


Figure 5. Block Diagram of the AW Compensation

According to engineering experience, the feedback gain K_a take for $1/K_p$ generally, can be variable in the range of $K_p/3 \sim 3K_p$ [11].

3.3. Speed Controller Design for PMSM

The effective plant as seen by speed controller is consisted by (2) and (3). A speed controller with excellent performance can be synthesized from 2DOF PI controller and AW compensation. The proposed speed control system of PMSM is shown in Figure 6.

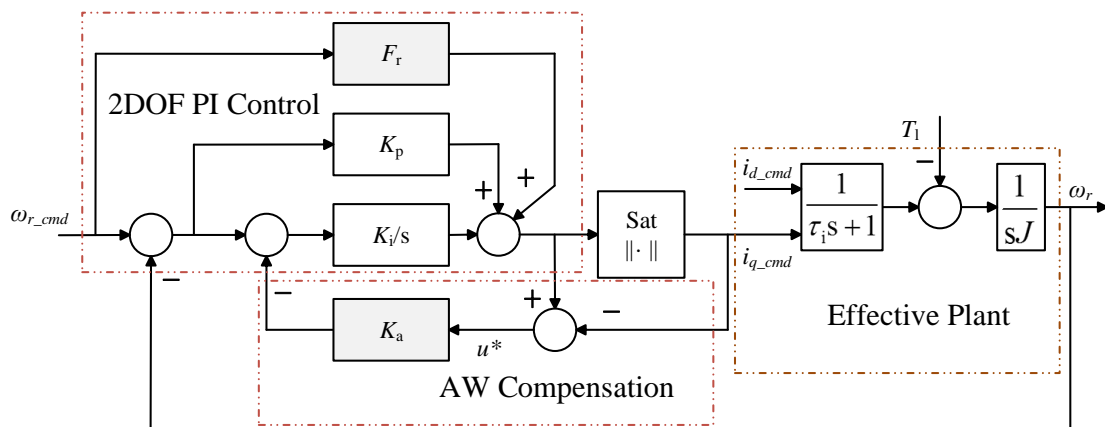


Figure 6. The Proposed Speed Control System of PMSM

In normal operation, the close-loop transfer function from T_1 to ω_r is

$$G_N(s) = -\frac{\tau_i s^2 + s}{J\tau_i s^3 + Js^2 + K_p s + K_i} \quad (8)$$

Parameter selection of the PI controller is count for much to the transient load-torque rejection. We chose K_p and K_i as follows

$$\begin{cases} K_p = J\alpha \\ K_i = J\left(\frac{\alpha}{2\zeta}\right)^2 \end{cases} \quad (9)$$

Where α is the bandwidth of the controller in pure P control, ζ is the relative damping in standard PI control [4].

To decrease the overshoot appearing in the step response of the standard 1DOF PI control, first-order dynamics can be selected as the desired transfer function of $G_x(s)$

$$G_X(s) = \frac{m\alpha}{s+m\alpha} \quad (10)$$

$G_X(s)$ has a pole at $s=-m\alpha$ and parameter m can be used to freely select the location of the pole. Then the feedforward controller can be obtained by (2), (3), (7), (9) and (10)

$$F_r(s) = J\alpha \frac{m\tau_i s^2 + (m-1)s - \frac{\alpha}{4\zeta^2}}{s+m\alpha} \quad (11)$$

The influence of parameter m to the close-loop system is shown in Figure 7 by bode diagram (with $\alpha=0.5$ and m varies from 0.2 to 2). From the bode diagram of $G_X(s)$ we can see that increases as the parameter m , the bandwidth of the system increases.

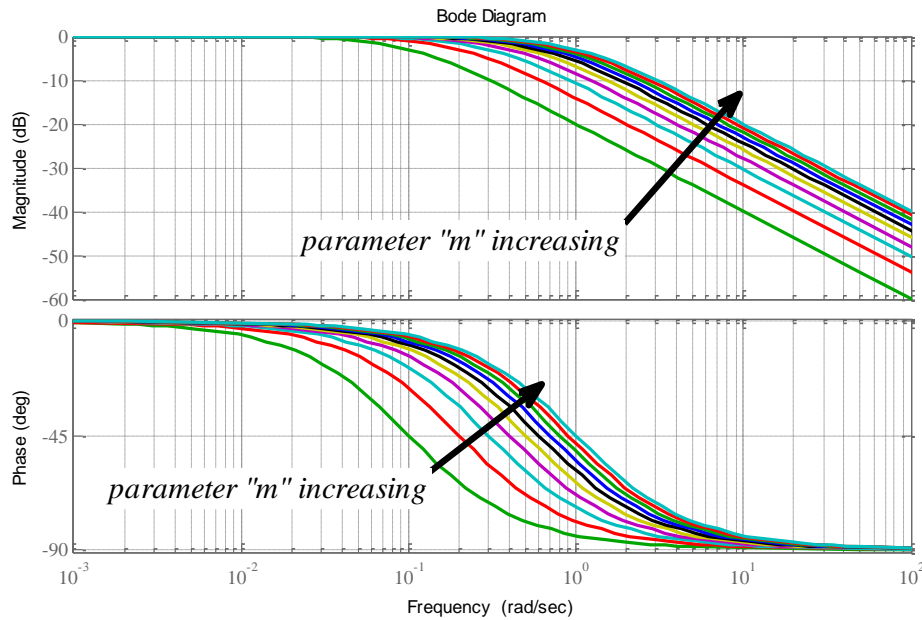


Figure 7. Bode Diagram of $G_X(s)$ [with $\alpha=0.5$ and m Varies from 0.2 to 2]

A special case is $m=1$, then

$$F_r(s) = J\alpha \frac{\tau_i s^2 - \frac{\alpha}{4\zeta^2}}{s+\alpha} \quad (12)$$

It can be further simplified in case of τ_i is very small, becomes simply a first-order low-pass filter

$$F_r(s) = -\frac{J \left(\frac{\alpha}{2\zeta} \right)^2}{s+\alpha} \quad (13)$$

When a saturation event is occurs ($u^* \neq 0$), the AW compensator will take action. Then the signal u^* will soon be reduced to zero. We choose $K_a = 1/K_p$ in preliminary design.

3.4. Digital Implementation of the Controller

The proposed speed controller can be implemented digitally on a DSP. Common discretization methods are summarized in Table 1.

Table 1. Discretization Methods and Transfer Relations

Forward Euler	Backward Euler	Trapezoidal(Bilinear)
$s=(z-1)/T_s$	$s=(z-1)/(zT_s)$	$s=[2(z-1)]/[T_s(z+1)]$

Where T_s is the sampling period. Among these methods, Trapezoidal method usually has better performance. With the usage of the transfer relation of Trapezoidal method, the transfer functions of the PI controller and the feedforward controller will become

$$F(z) = K_p + K_i \frac{T_s}{2} \frac{1+z^{-1}}{1-z^{-1}} \quad (14)$$

$$F_i(z) = -\frac{J \left(\frac{\alpha}{2\zeta} \right)^2}{\frac{2}{T_s} \frac{1-z^{-1}}{1+z^{-1}} + \alpha} \quad (15)$$

4. Simulations and Experiments

4.1. Simulations

For simulation tests, the control system of PMSM which is shown in Figure 1 is established in MATLAB/Simulink. Then we compare the performance of standard speed controller and the proposed speed controller. The inertia of the motor used in simulation is $J=0.4\text{kg} \cdot \text{m}^2$, the time constant of the inner loop is about $\tau_i=0.05\text{s}$, the limit of i_{q_cmd} is 7.6A.

Simulation condition 1: with no load, speed reference 500rpm is given at $t=0\text{s}$, and 200rpm at $t=5\text{s}$.

The performance of the AW compensation method is shown in Figure 8 and Figure 9. When the step speed reference comes, i_{q_cmd} quickly rise to the limit and hold on. Then the “windup” problem comes. Both of the speed and current responses of the control system without the AW compensation show great overshoot which is more than 55% under step speed references. Furthermore, the overshoot lead to the reverse impact on i_{q_cmd} . The responses of the proposed controller shows better performance. The integrator quickly out of saturation so that the overshoot is smaller, about 16%.

Simulation condition 2: with no load, speed reference 500rpm is given at $t=0\text{s}$, then load-torque $T_l=8\text{Nm}$ is given at $t=7\text{s}$, and speed reference 200rpm at $t=10\text{s}$.

The performance of the 2DOF PI control method is shown in Figure 10. By the usage of a standard PI control method, good load-torque rejection can be met when we choose $K_p=0.2$ and $K_i=0.3$ (with $\alpha=0.5$). Then we use a feedforward controller designed according to (15), the speed-reference tracking is going to be better and good load-torque rejection still remains.

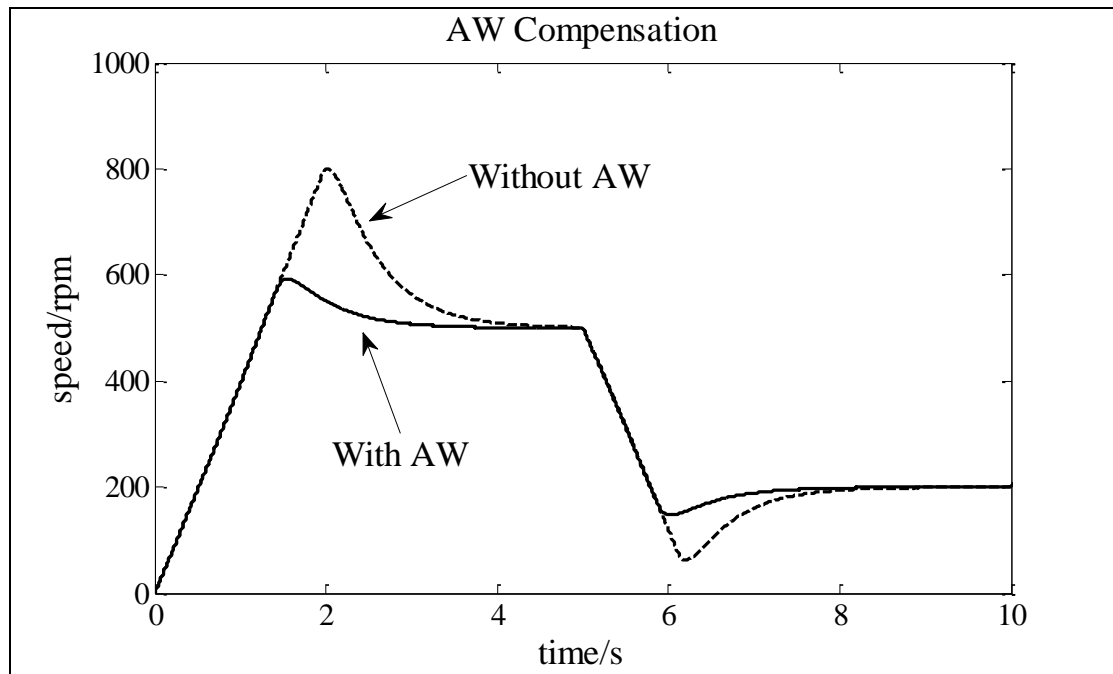


Figure 8. The Performance of the AW Compensation Method [speed]

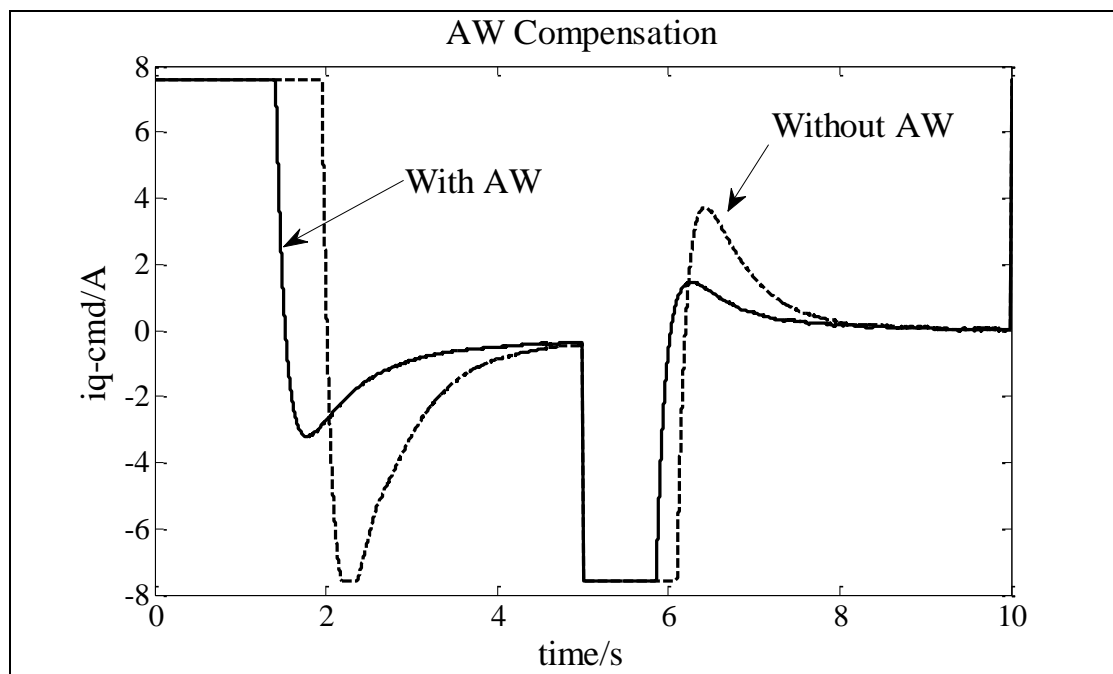


Figure 9. The Performance of the AW Compensation Method [current]

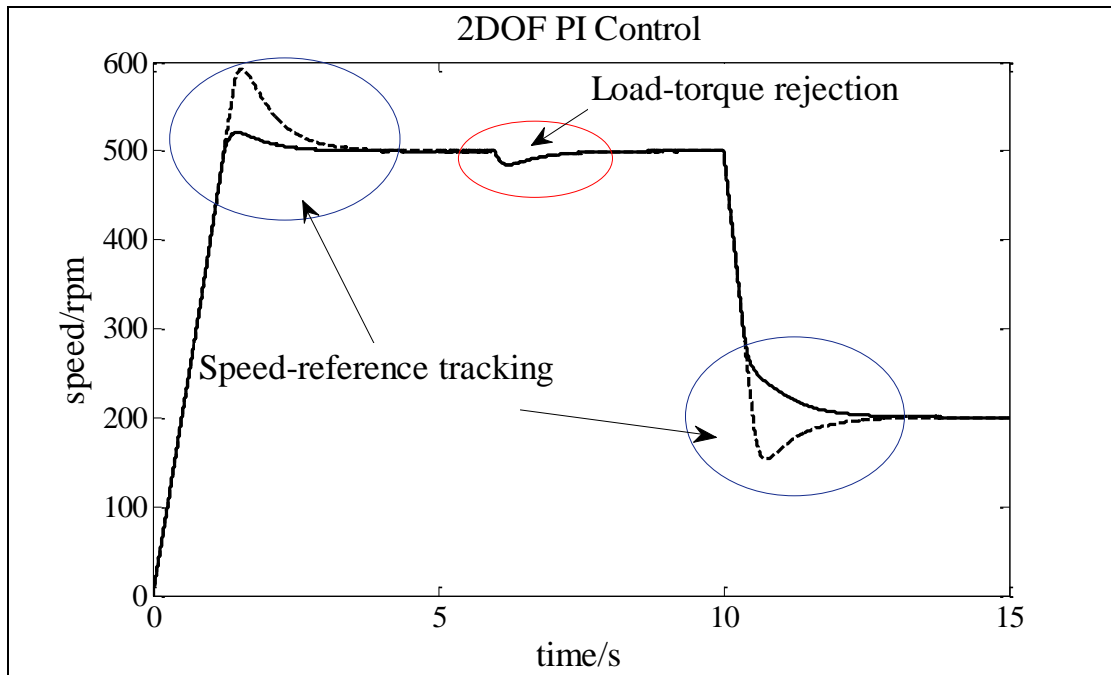


Figure 10. The Performance of the 2DOF PI Control Method

4.2. Experiments

For experimental tests, a drive system of PMSM which is shown in Figure 11 is established. A magnetic powder brake works as the load of PMSM. The inertia of the motor used in experiment is $J=0.4\text{kg} \cdot \text{m}^2$, the time constant of the inner loop is about $\tau_i=0.05\text{s}$, the limit of i_{q_cmd} is 7.6A.

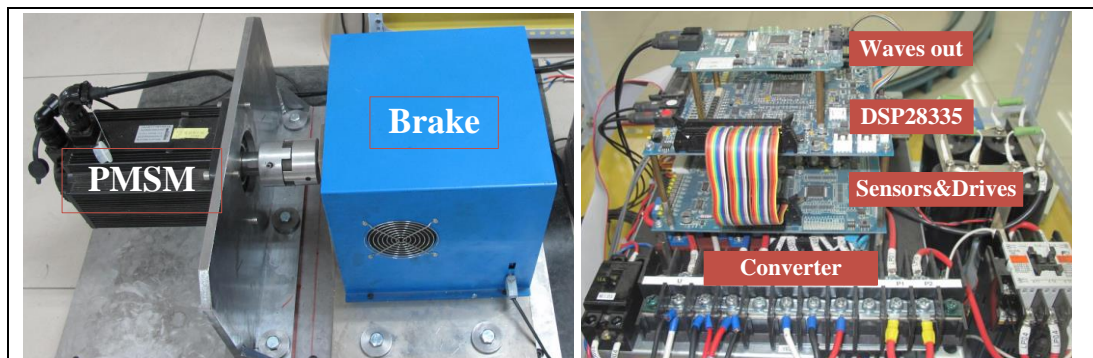


Figure 11. The Setups of the PMSM Drive System

The performance of the proposed control method is shown in Figure 12 to Figure 14.

Experimental condition 1: With no load, speed reference 500rpm is given at $t=0.5\text{s}$. The speed responses are shown in Figure 12. The response of the controller with AW compensation shows better performance. By the use of AW compensation method, the overshoot is reduced from 35% to 15%.

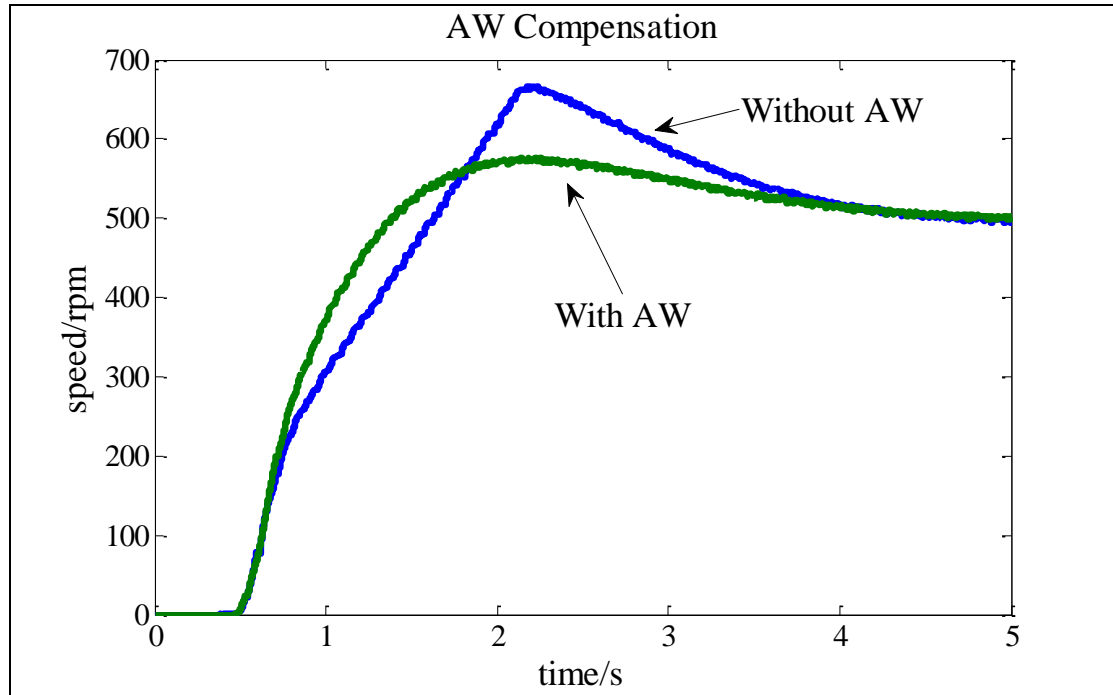
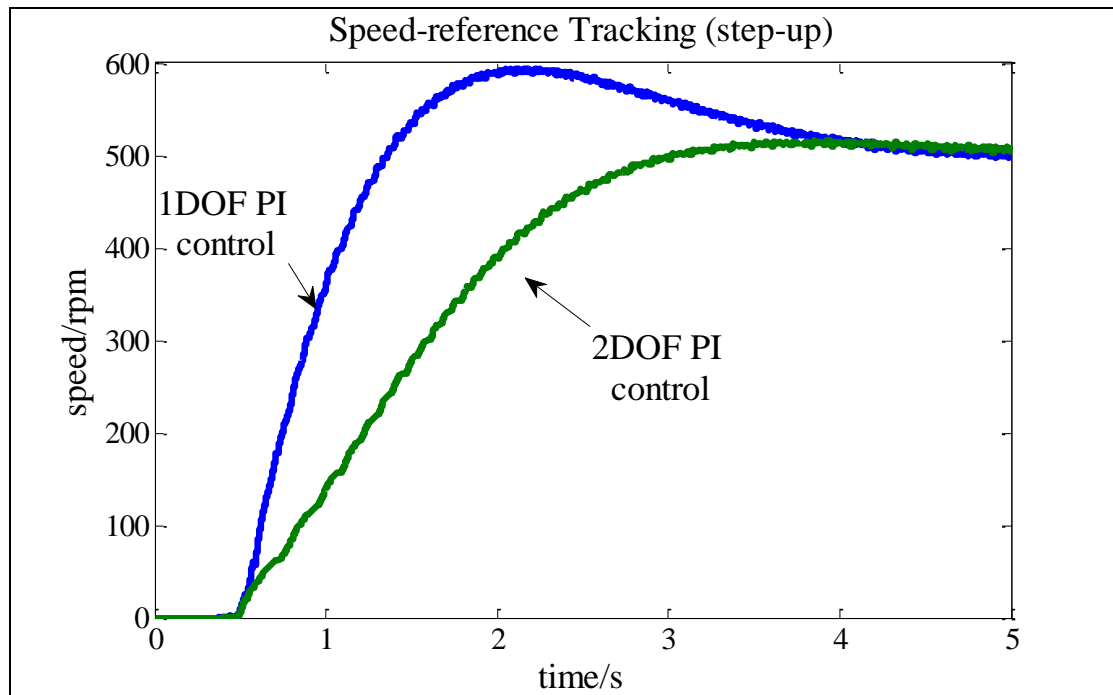
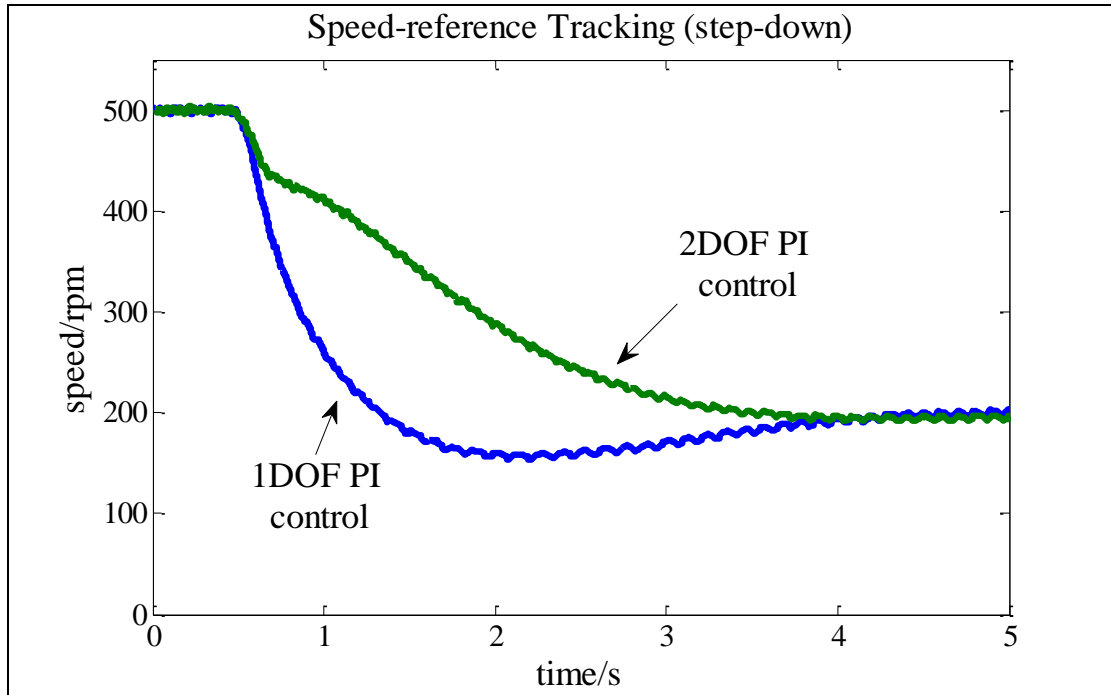


Figure 12. The Speed Responses with AW and Without AW Compensation

Experimental condition 2: With no load, speed reference step-up (from 0rpm to 500rpm) and step-down (from 500rpm to 200rpm) are given; Figure 13 shows the speed-reference tracking performance of the system.



(a) Step-up of speed reference



(b) Step-down of speed reference

Figure 13. The Speed-Reference Tracking Performance

Experimental condition 3: At steady operation of 500rpm, load-torque 8Nm (there may be a little error) is given at $t=0.5s$. Figure 14 shows the load-torque performance of the system.

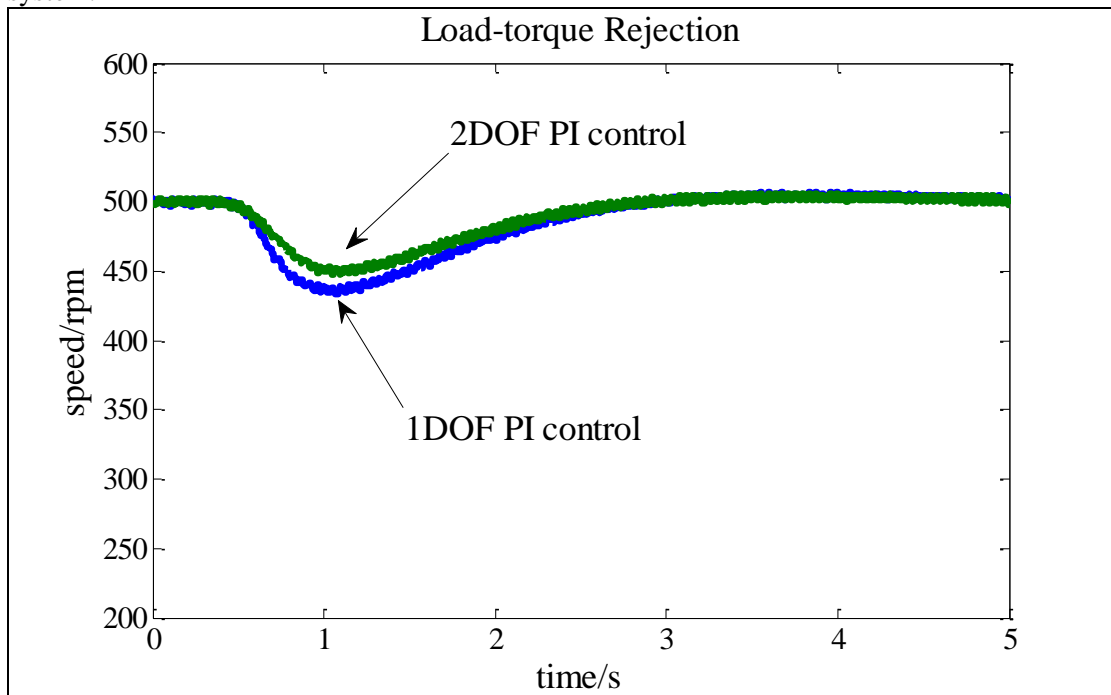


Figure 14. The Load-Torque Rejection Performance

With the usage of 2DOF PI control method, the speed-reference tracking performance has improved a lot, while the load-torque rejection is remain almost the same as in the case of the 1DOF PI control. In other words, the speed-reference

tacking performance and the load-torque rejection performance can be adjusted separately. The experimental results shows that the proposed controller gives good reference tracking as well as good load rejection.

5. Conclusion

In this paper, a novel speed control method of PMSM has been proposed. The proposed controller can be easily implemented in software and needs no additional hardware or sensor. The “back calculation and tracking” AW compensation method can easily solve the “windup” problem of the controller. The 2DOF method gives the controller the ability to meet good speed-reference tracking and good load-torque rejection simultaneously. The effectiveness and superiority of the proposed method has been verified by the simulation and experimental results.

Hence, the load of the motor in real applications may not be as simple as in this paper. More advanced speed control schemes may be useful if the mechanical systems are more complex, such as a multibody system, or the nonlinearities of the model cannot be linearized. In that case, a higher order model of the load and the motor will be needed for stability analysis.

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